

平面応力問題

弾塑性力学

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第6回: 2014/6/26

片持ちはりにおける応力・ひずみ

5次の多項式によるアエリーの応力関数

$$F(x, z) = {}^2F(x, z) + {}^3F(x, z) + {}^4F(x, z) + {}^5F(x, z)$$

定数の未知数 (15個)

$$a_2, b_2, c_2, a_3, b_3, c_3, d_3, a_4, b_4, c_4, d_4, a_5, b_5, c_5, d_5$$

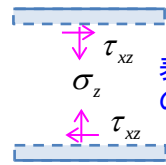
$$F(x, z) := a_2 x^2 + b_2 xz + c_2 z^2 + a_3 x^3 + b_3 x^2 z + c_3 xz^2 + d_3 z^3 + a_4 x^4 + b_4 x^3 z + c_4 x^2 z^2 + d_4 xz^3 \dots$$

$$+ \left\{ -a_4 - \frac{1}{3}c_4 \right\} z^4 + a_5 x^5 + b_5 x^4 z + c_5 x^3 z^2 + d_5 x^2 z^3 + \left\{ -5a_5 - c_5 \right\} xz^4 + \left\{ -\frac{1}{5}b_5 - \frac{1}{5}d_5 \right\} z^5$$

境界における応力条件

$$\sigma_z dx = 0, \tau_{xz} dx = 0$$

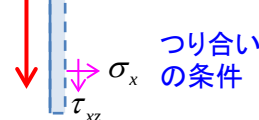
$$\therefore \sigma_z = 0, \tau_{xz} = 0$$



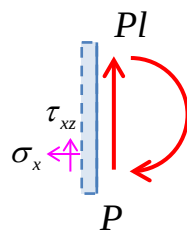
表面応力の条件

$$\sigma_x dz = 0$$

$$\therefore \sigma_x = 0$$



つり合いの条件



応力成分

$$\begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial^2 F}{\partial z^2} \\ \frac{\partial^2 F}{\partial x^2} \\ -\frac{\partial^2 F}{\partial x \partial z} \end{Bmatrix} = {}^2 \begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix} + {}^3 \begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix} + {}^4 \begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix} + {}^5 \begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix}$$

$$\sigma_x(x, z) = 2c_5 x^3 + \{2c_4 + 6d_5 z\} x^2 + \left\{ \{-60a_5 - 12c_5\} z^2 + 6d_4 z + 2c_3 \right\} x \dots$$

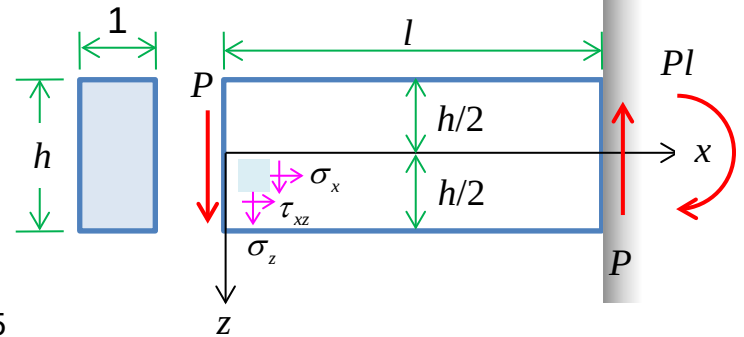
$$+ \{-4b_5 - 4d_5\} z^3 + \{-12a_4 - 4c_4\} z^2 + 6d_3 z + 2c_2$$

$$\sigma_z(x, z) = 20a_5 x^3 + \{12a_4 + 12b_5 z\} x^2 + \{6b_4 z + 6c_5 z^2 + 6a_3\} x \dots$$

$$+ 2a_2 + 2c_4 z^2 + 2b_3 z + 2d_5 z^3$$

$$\tau_{xz}(x, z) = 4b_5 x^3 + \{-3b_4 - 6c_5 z\} x^2 + \{-4c_4 z - 6d_5 z^2 - 2b_3\} x \dots$$

$$+ \left\{ 20a_5 + 4c_5 \right\} z^3 - 3d_4 z^2 - 2c_3 z - b_2$$



$$z = -h/2 \Rightarrow \sigma_z = 0, \tau_{xz} = 0$$

$$z = h/2 \Rightarrow \sigma_z = 0, \tau_{xz} = 0$$

強形式

弱形式

$$x = 0 \Rightarrow \sigma_x = 0, \int_{-h/2}^{h/2} \tau_{xz} dz + P = 0$$

$$x = l \Rightarrow \int_{-h/2}^{h/2} \tau_{xz} dz + P = 0$$

$$\int_{-h/2}^{h/2} \sigma_x z dz + Pl = 0$$

弱形式

強形式

$$\left[\begin{array}{l} \sigma_x(0, z) = 0 \text{ collect, } x, z \rightarrow \{-4 \cdot b_5 - 4 \cdot d_5\} z^3 + \{-12 \cdot a_4 - 4 \cdot c_4\} z^2 + 6 \cdot d_3 z + 2 \cdot c_2 = 0 \\ \sigma_z\left(x, \frac{-h}{2}\right) = 0 \text{ collect, } x, z \rightarrow 20 \cdot a_5 x^3 + \{12 \cdot a_4 - 6 \cdot b_5 h\} x^2 + \left\{-3 \cdot b_4 h + \frac{3}{2} \cdot c_5 h^2 + 6 \cdot a_3\right\} x + 2 \cdot a_2 + \frac{1}{2} \cdot c_4 h^2 - b_3 h - \frac{1}{4} \cdot d_5 h^3 = 0 \\ \tau_{xz}\left(x, \frac{-h}{2}\right) = 0 \text{ collect, } x, z \rightarrow -4 \cdot b_5 x^3 + \{-3 \cdot b_4 + 3 \cdot c_5 h\} x^2 + \left\{2 \cdot c_4 h - \frac{3}{2} \cdot d_5 h^2 - 2 \cdot b_3\right\} x - \frac{1}{8} \cdot \{20 \cdot a_5 + 4 \cdot c_5\} \cdot h^3 - \frac{3}{4} \cdot d_4 h^2 + c_3 h - b_2 = 0 \\ \sigma_z\left(x, \frac{h}{2}\right) = 0 \text{ collect, } x, z \rightarrow 20 \cdot a_5 x^3 + \{12 \cdot a_4 + 6 \cdot b_5 h\} x^2 + \left\{3 \cdot b_4 h + \frac{3}{2} \cdot c_5 h^2 + 6 \cdot a_3\right\} x + 2 \cdot a_2 + \frac{1}{2} \cdot c_4 h^2 + b_3 h + \frac{1}{4} \cdot d_5 h^3 = 0 \\ \tau_{xz}\left(x, \frac{h}{2}\right) = 0 \text{ collect, } x, z \rightarrow -4 \cdot b_5 x^3 + \{-3 \cdot b_4 - 3 \cdot c_5 h\} x^2 + \left\{-2 \cdot c_4 h - \frac{3}{2} \cdot d_5 h^2 - 2 \cdot b_3\right\} x + \frac{1}{8} \cdot \{20 \cdot a_5 + 4 \cdot c_5\} \cdot h^3 - \frac{3}{4} \cdot d_4 h^2 - c_3 h - b_2 = 0 \end{array} \right]$$

連立式

$$\left[\begin{array}{l} -4 \cdot b_5 - 4 \cdot d_5 = 0 \quad -12 \cdot a_4 - 4 \cdot c_4 = 0 \quad 6 \cdot d_3 = 0 \quad 2 \cdot c_2 = 0 \\ 20 \cdot a_5 = 0 \quad 12 \cdot a_4 - 6 \cdot b_5 h = 0 \quad -3 \cdot b_4 h + \frac{3}{2} \cdot h^2 \cdot c_5 + 6 \cdot a_3 = 0 \quad 2 \cdot a_2 + \frac{1}{2} \cdot h^2 \cdot c_4 - b_3 h - \frac{1}{4} \cdot h^3 \cdot d_5 = 0 \\ -4 \cdot b_5 = 0 \quad -3 \cdot b_4 + 3 \cdot c_5 h = 0 \quad 2 \cdot c_4 h - \frac{3}{2} \cdot d_5 h^2 - 2 \cdot b_3 = 0 \quad -\frac{1}{8} \cdot \{20 \cdot a_5 + 4 \cdot c_5\} \cdot h^3 - \frac{3}{4} \cdot d_4 h^2 + c_3 h - b_2 = 0 \\ 20 \cdot a_5 = 0 \quad 12 \cdot a_4 + 6 \cdot b_5 h = 0 \quad 3 \cdot b_4 h + \frac{3}{2} \cdot h^2 \cdot c_5 + 6 \cdot a_3 = 0 \quad 2 \cdot a_2 + \frac{1}{2} \cdot h^2 \cdot c_4 + b_3 h + \frac{1}{4} \cdot h^3 \cdot d_5 = 0 \\ -4 \cdot b_5 = 0 \quad -3 \cdot b_4 - 3 \cdot c_5 h = 0 \quad -2 \cdot c_4 h - \frac{3}{2} \cdot d_5 h^2 - 2 \cdot b_3 = 0 \quad \frac{1}{8} \cdot \{20 \cdot a_5 + 4 \cdot c_5\} \cdot h^3 - \frac{3}{4} \cdot d_4 h^2 - c_3 h - b_2 = 0 \end{array} \right]$$

線形代数による解

$$\begin{aligned} b_2 &= -\frac{3h^2}{4} d_4 \\ a_2 &= c_2 = 0 \\ a_3 &= b_3 = c_3 = d_3 = 0 \\ a_4 &= b_4 = c_4 = 0 \\ a_5 &= b_5 = c_5 = d_5 = 0 \end{aligned}$$

弱形式

$$\left. \begin{array}{l} F(x, z) = d_4 x z \left(z^2 - \frac{3h^2}{4} \right) \\ \left\{ \begin{array}{l} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{array} \right\} = \left\{ \begin{array}{l} 6d_4 x z \\ 0 \\ -\frac{3}{4} d_4 (2z - h)(2z + h) \end{array} \right\} \end{array} \right\} \Rightarrow \left. \begin{array}{l} \int_{-h/2}^{h/2} \tau_{xz}(0, z) dz + P = 0 \\ \int_{-h/2}^{h/2} \tau_{xz}(l, z) dz + P = 0 \\ \int_{-h/2}^{h/2} \sigma_x z dz + Pl = 0 \end{array} \right\} \Rightarrow \text{どちらも } d_4 = -\frac{2P}{h^3} \Rightarrow F(x, z) = \frac{Pxz(3h^2 - 4z^2)}{2h^3}$$

エアリーの応力関数

応力成分の解析解

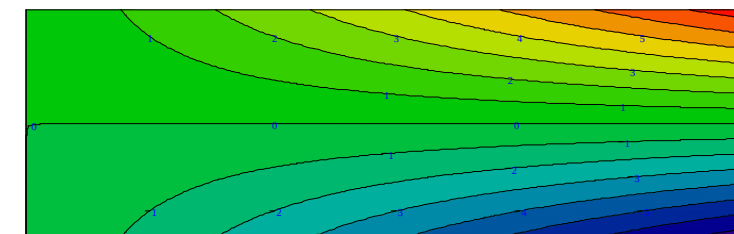
$$\begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix} = \begin{Bmatrix} \frac{\partial^2 F}{\partial z^2} \\ \frac{\partial^2 F}{\partial x^2} \\ -\frac{\partial^2 F}{\partial x \partial z} \end{Bmatrix} = \begin{Bmatrix} -\frac{12P}{h^3}xz \\ 0 \\ -\frac{6P}{h^3}\left(\left(\frac{h}{2}\right)^2 - z^2\right) \end{Bmatrix}$$



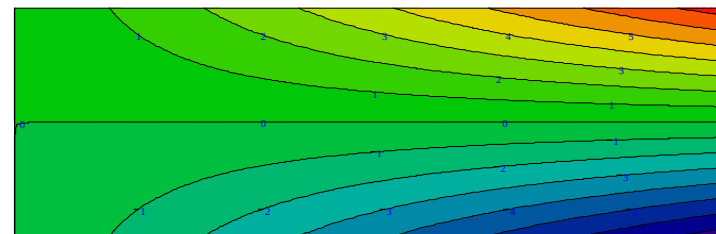
平面応力条件下のひずみ成分

$$\begin{Bmatrix} \varepsilon_x \\ \varepsilon_z \\ \gamma_{xz} \end{Bmatrix} = \frac{1}{E} \begin{bmatrix} 1 & -\nu & 0 \\ -\nu & 1 & 0 \\ 0 & 0 & 2(1+\nu) \end{bmatrix} \begin{Bmatrix} \sigma_x \\ \sigma_z \\ \tau_{xz} \end{Bmatrix}$$

$$l/h=5/2, \nu=1/3$$



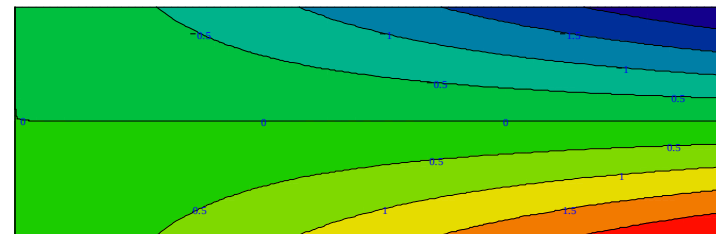
σ_x/P



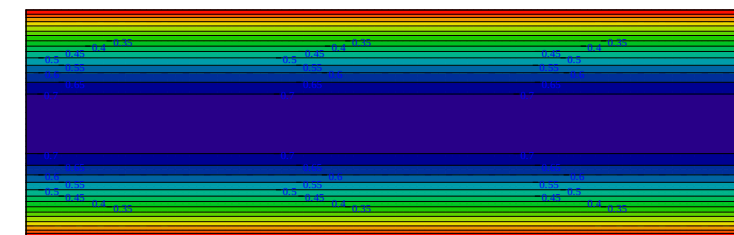
$E\varepsilon_x/P$



σ_z/P

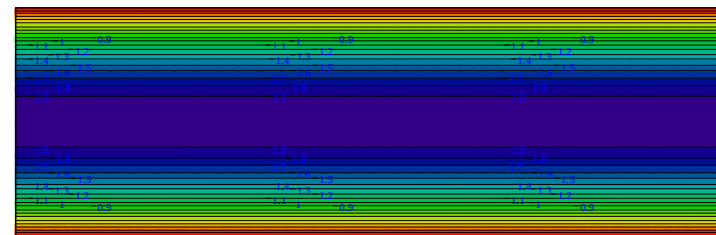


$E\varepsilon_z/P$



τ_{xz}/P

応力分布



$E\gamma_{xz}/P$

ひずみ分布

円孔をもつ無限平板の引っ張り問題

境界条件① $r = \infty \Rightarrow \sigma_{xx} = \sigma_0, \sigma_{yy} = k\sigma_0, \tau_{xy} = 0$

境界条件② $r = a \Rightarrow \sigma_{rr} = 0, \tau_{r\theta} = 0$

$$\boldsymbol{\sigma}|_{r=\infty} = \begin{bmatrix} \sigma_{rr} & \tau_{r\theta} \\ \tau_{r\theta} & \sigma_{\theta\theta} \end{bmatrix}_{r=\infty} = \mathbf{R} \cdot \begin{bmatrix} \sigma_{x0} & 0 \\ 0 & \sigma_{y0} \end{bmatrix} \cdot \mathbf{R}^T$$

$$= p_0 \underbrace{\begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}}_{\text{等方}} + q_0 \underbrace{\begin{bmatrix} \cos 2\theta & -\sin 2\theta \\ -\sin 2\theta & -\cos 2\theta \end{bmatrix}}_{\text{偏差}}$$

$$= {}^1\boldsymbol{\sigma}|_{r=\infty} + {}^2\boldsymbol{\sigma}|_{r=\infty}$$

$$p_0 = \frac{\sigma_{x0} + \sigma_{y0}}{2} = \frac{1+k}{2} \sigma_{x0}$$

$$q_0 = \frac{\sigma_{x0} - \sigma_{y0}}{2} = \frac{1-k}{2} \sigma_{x0}$$

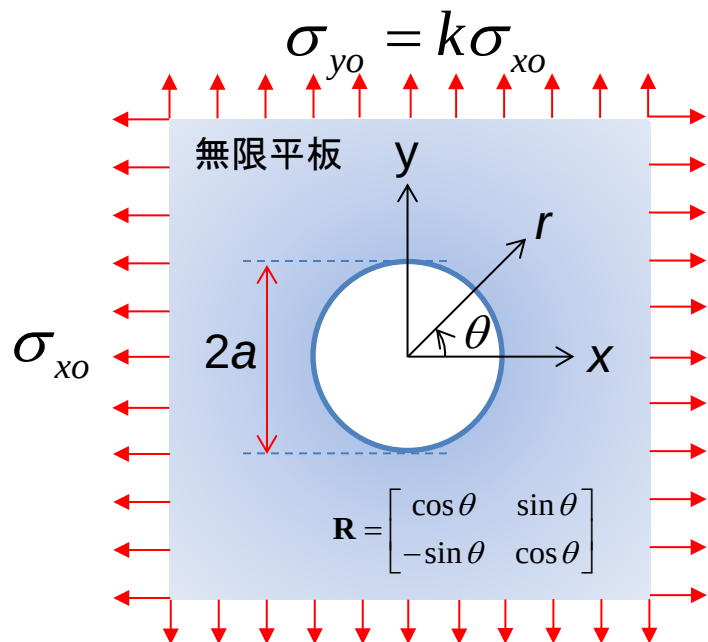
$$\boldsymbol{\sigma}|_{r=a} = \begin{bmatrix} 0 & 0 \\ 0 & \sigma_{\theta\theta}|_{r=a} \end{bmatrix} = {}^1\boldsymbol{\sigma}|_{r=a} + {}^2\boldsymbol{\sigma}|_{r=a}$$

第1系

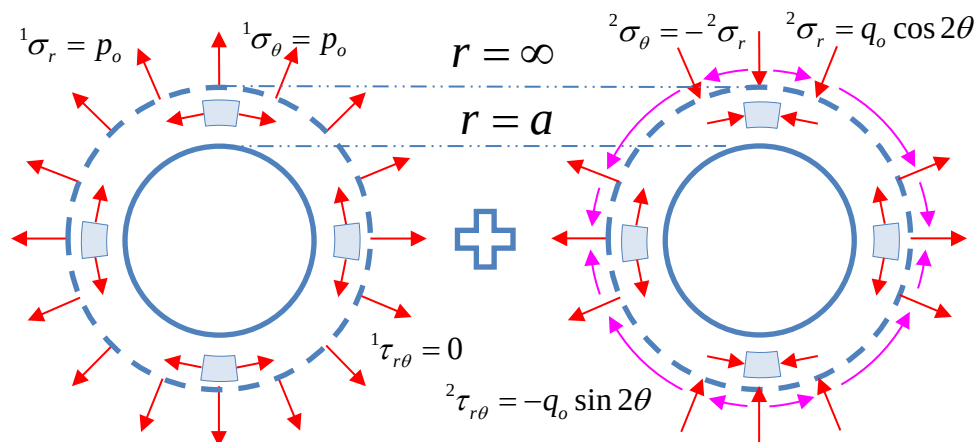
$${}^1\boldsymbol{\sigma} = \begin{bmatrix} {}^1\sigma_{rr} & {}^1\tau_{r\theta} \\ {}^1\tau_{r\theta} & {}^1\sigma_{\theta\theta} \end{bmatrix}$$

第2系

$${}^2\boldsymbol{\sigma} = \begin{bmatrix} {}^2\sigma_{rr} & {}^2\tau_{r\theta} \\ {}^2\tau_{r\theta} & {}^2\sigma_{\theta\theta} \end{bmatrix}$$



重ね合わせ



第1系(等方)

第2系(偏差)

第1系(等方)

$${}^1\boldsymbol{\sigma}\Big|_{r=\infty} = p_o \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

↓ 応力関数

$${}^1F(r, \theta) = A_{01}r^2 + A_{02}r^2\theta + A_{03}\ln r + A_{04}\theta$$

↓ 表面応力

$$\begin{aligned} {}^1\sigma_{rr}(r, \theta) &= 2A_{01} + 2A_{02}\theta + A_{03}r^{-2} \\ {}^1\sigma_{\theta\theta}(r, \theta) &= 2A_{01} + 2A_{02}\theta - A_{03}r^{-2} \\ {}^1\tau_{r\theta}(r, \theta) &= -A_{02} + A_{04}r^{-2} \end{aligned}$$

↓

$$\begin{aligned} {}^1\sigma_{rr}(r, \theta) &= p_o \left(1 - (a/r)^2\right) \\ {}^1\sigma_{\theta\theta}(r, \theta) &= p_o \left(1 + (a/r)^2\right) \\ {}^1\tau_{r\theta}(r, \theta) &= 0 \end{aligned}$$

$$\begin{aligned} {}^1\sigma_{rr}\Big|_{r=\infty} &= p_o \\ {}^1\tau_{r\theta}\Big|_{r=\infty} &= 0 \\ {}^1\sigma_{rr}\Big|_{r=a} &= 0 \\ {}^1\tau_{r\theta}\Big|_{r=a} &= 0 \end{aligned}$$

$$\begin{aligned} A_{01} &= p_o/2 \\ A_{02} &= 0 \\ A_{03} &= -a_o^2 p_o \\ A_{04} &= 0 \end{aligned}$$

第2系(偏差)

$${}^2\boldsymbol{\sigma}\Big|_{r=\infty} = q_o \begin{bmatrix} \cos 2\theta & -\sin 2\theta \\ -\sin 2\theta & -\cos 2\theta \end{bmatrix}$$

↓ 応力関数

$${}^2F(r, \theta) = \left(A_{21}r^4 + A_{22} + A_{23}r^2 + A_{24}r^{-2} \right) \cos 2\theta$$

↓ 表面応力

$$\begin{aligned} {}^2\sigma_{rr}(r, \theta) &= -2 \left(2A_{22}r^{-2} + A_{23} + 3A_{24}r^{-4} \right) \cos 2\theta \\ {}^2\sigma_{\theta\theta}(r, \theta) &= -2 \left(6A_{21}r^2 + A_{23} + 3A_{24}r^{-4} \right) \cos 2\theta \\ {}^2\tau_{r\theta}(r, \theta) &= 2 \left(3A_{21}r^2 + A_{22}r^{-2} + A_{23} - 3A_{24}r^{-4} \right) \sin 2\theta \end{aligned}$$

↓

$$\begin{aligned} {}^2\sigma_{rr}(r, \theta) &= q_o \left(1 - (a/r)^2\right) \left(1 - 3(a/r)^2\right) \cos 2\theta \\ {}^2\sigma_{\theta\theta}(r, \theta) &= -q_o \left(1 + 3(a/r)^2\right) \cos 2\theta \\ {}^2\tau_{r\theta}(r, \theta) &= -q_o \left(1 - (a/r)^2\right) \left(1 + 3(a/r)^2\right) \sin 2\theta \end{aligned}$$

$$\begin{aligned} {}^2\sigma_{rr}\Big|_{r=\infty} &= q_o \cos 2\theta \\ {}^2\tau_{r\theta}\Big|_{r=\infty} &= -q_o \sin 2\theta \\ {}^2\sigma_{rr}\Big|_{r=a} &= 0 \\ {}^2\tau_{r\theta}\Big|_{r=a} &= 0 \end{aligned}$$

$$\begin{aligned} A_{21} &= 0 \\ A_{22} &= q_o a^2 \\ A_{23} &= -q_o/2 \\ A_{24} &= -q_o a^4/2 \end{aligned}$$

重ね合わせ：第1系+第2系 $\sigma = {}^1\sigma + {}^2\sigma$

$$p_o = \frac{1+k}{2} \sigma_{x_o}$$

応力解

$$\begin{aligned} \sigma_{rr}(r, \theta)/p_o &= \left(1 - (a/r)^2\right) + (q_o/p_o) \left(1 - (a/r)^2\right) \left(1 - 3(a/r)^2\right) \cos 2\theta \\ \sigma_{\theta\theta}(r, \theta)/p_o &= \left(1 + (a/r)^2\right) - (q_o/p_o) \left(1 + 3(a/r)^2\right) \cos 2\theta \\ \tau_{r\theta}(r, \theta)/p_o &= -(q_o/p_o) \left(1 - (a/r)^2\right) \left(1 + 3(a/r)^2\right) \sin 2\theta \end{aligned}$$

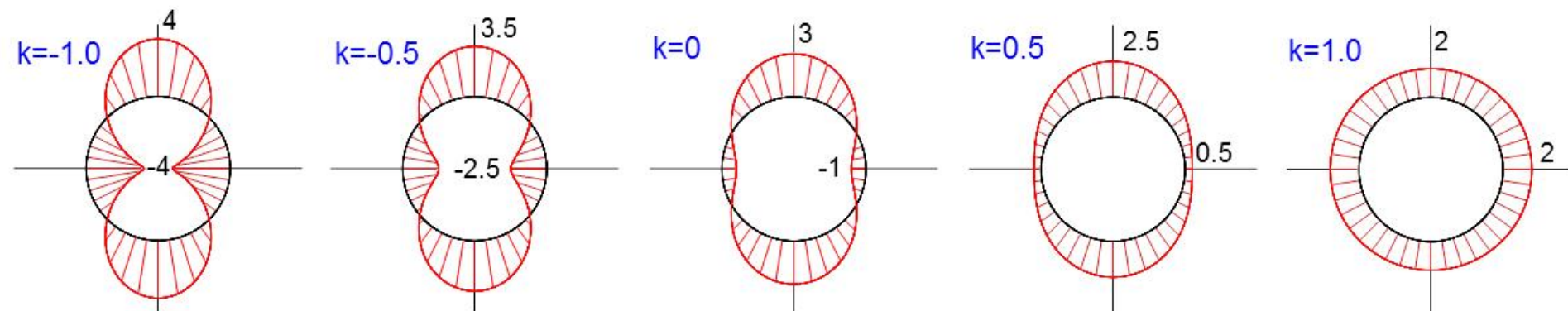
$$r = a$$

孔縁での最大応力 応力集中係数

$$\begin{aligned} \sigma_{rr}(a, \theta)/p_o &= 0 \\ \sigma_{\theta\theta}(a, \theta)/p_o &= 2 - 4 \frac{1-k}{1+k} \cos 2\theta \\ \tau_{r\theta}(a, \theta)/p_o &= 0 \end{aligned}$$

$$\begin{aligned} \frac{\sigma_{x_{\max}}}{\sigma_{x_o}} &= \frac{\sigma_{\theta}(a, \pi/2)}{\sigma_{x_o}} = 3 - k \\ \frac{\sigma_{y_{\max}}}{\sigma_{x_o}} &= \frac{\sigma_{\theta}(a, 0)}{\sigma_{x_o}} = 3k - 1 \end{aligned}$$

週方向応力 $\sigma_{\theta\theta}/\sigma_{x_o}(r=a)$



円孔をもつ無限平板の引っ張り問題による応力集中係数

宿題(2013年試験問題)

図2に示す単純ばりの上面($z=-h/2$)に等分布荷重 w を受けている。重調和関数を満たすエアリー応力関数を決定するために、どのような応力の境界条件に求められているのか、その強形式および弱形式を書けよ。

