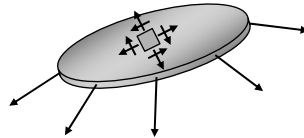


Stresses in Thin-Walled Pressure Vessels

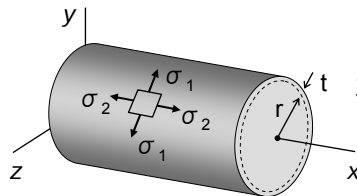
薄肉の圧力容器の応力

球形タンクや円筒形タンク, あるいはパイプライン

内圧としてかかる力は壁に対して直角と考えられる



内圧を受けるタンク: 内径が r , 板厚が t



σ_1 はhoop stressと呼ばれる
木樽の締め付けバンドに作用する力のように

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Stresses in Thin-Walled Pressure Vessels

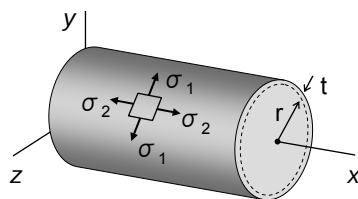
薄肉の圧力容器の応力

Hoop stressを求める

$$\sum F_z = 0$$

$$\sigma_1(2t \cdot \Delta x) - p(2r \cdot \Delta x) = 0$$

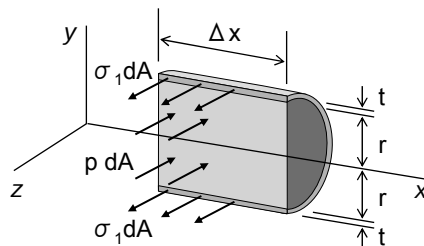
$$\sigma_1 = \frac{pr}{t}$$



$$\sum F_x = 0$$

$$\sigma_2(2\pi r t) - p(\pi r^2) = 0$$

$$\sigma_2 = \frac{pr}{2t}$$



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Stresses in Thin-Walled Pressure Vessels

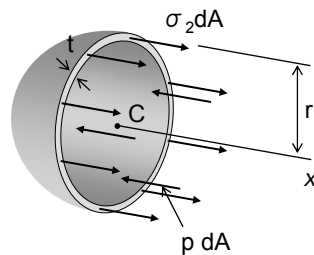
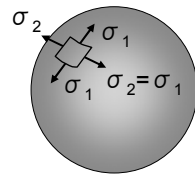
薄肉の圧力容器の応力

Hoop stressを求める

$$\sum F_x = 0$$

$$\sigma_2(2\pi r t) - p(\pi r^2) = 0$$

$$\sigma_1 = \sigma_2 = \frac{pr}{2t}$$



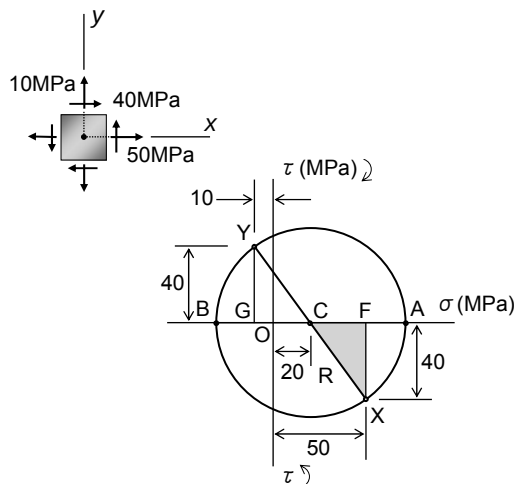
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Mohr's Circle for Plane Stress

平面応力下のモールの応力円

Example 7.02

(a) Example 7.01に対してMohrの応力円を描く



$$\begin{aligned}\sigma_{ave} &= \frac{\sigma_x + \sigma_y}{2} \\ &= \frac{50 + (-10)}{2} = 20 \text{ MPa}\end{aligned}$$

三角形CFXで

$$CF = 50 - 20 = 30 \text{ MPa}$$

$$FX = 40 \text{ MPa}$$

$$\begin{aligned}\therefore R = CX &= \sqrt{30^2 + 40^2} \\ &= 50 \text{ MPa}\end{aligned}$$

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Mohr's Circle for Plane Stress 平面応力下のモールの応力円

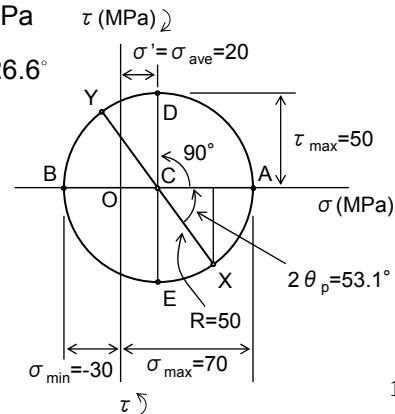
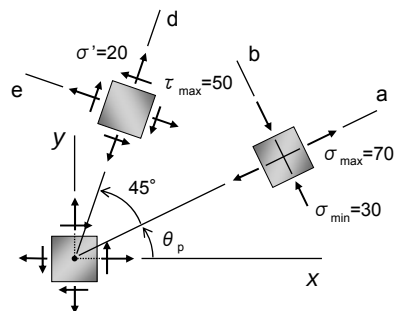
Example 7.02

(b) 主平面と主応力, (c) 最大せん断応力

$$\sigma_{\max} = OA = OC + CA = 20 + 50 = 70 \text{ MPa}$$

$$\sigma_{\min} = OB = OC - BC = 20 - 50 = -30 \text{ MPa}$$

$$\tan 2\theta_p = \frac{FX}{CF} = \frac{40}{30} \quad 2\theta_p = 53.1^\circ \quad \theta_p = 26.6^\circ$$



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Yield Criteria for Ductile Materials under Plane Stress 延性材料での降伏条件

Maximum Shearing Stress Criterion

Sec.1.11より, 引張試験での最大せん断応力は $\frac{1}{2}\sigma_Y$

Fracture Criteria for Brittle Materials under Plane Stress ぜい性材料での破壊条件

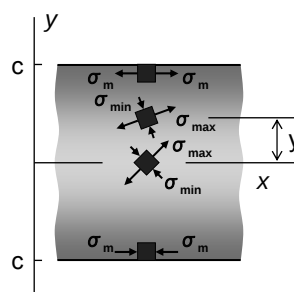
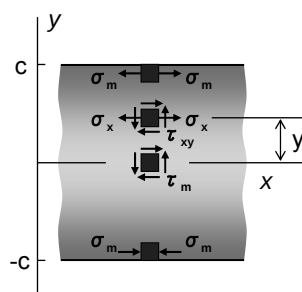
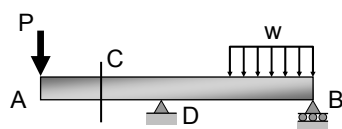
Maximum Normal Stress Criterion

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Chapter 8. Principal Stresses under Given Loading ある荷重下での主応力

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Principal Stresses in a Beam 梁の中の主応力

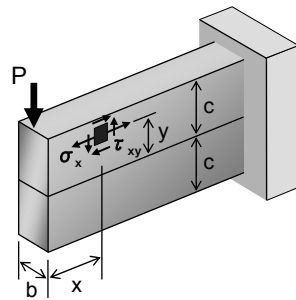


$$\sigma_x = -\frac{My}{I}$$

$$\tau_{xy} = -\frac{VQ}{It}$$

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Principal Stresses in a Beam 梁の中の主応力



$$I = \frac{bh^3}{12} = \frac{Ac^2}{3}$$

$$\sigma_x = \frac{Pxy}{I} = \frac{Pxy}{Ac^2/3} = \frac{3P}{A} \frac{xy}{c^2}$$

$$\tau_{xy} = \frac{3P}{2A} \left(1 - \frac{y^2}{c^2} \right)$$

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	$x=2c$		$x=8c$	
y/c	$\sigma_{\min}/\sigma_m$	$\sigma_{\max}/\sigma_m$	$\sigma_{\min}/\sigma_m$	$\sigma_{\max}/\sigma_m$
1.0	0		0	
0.8	-0.010		-0.001	
0.6	-0.040		-0.003	
0.4	-0.090		-0.007	
0.2	-0.160		-0.017	
0.0	-0.250		-0.063	
-0.2	-0.360		-0.217	
-0.4	-0.490		-0.407	
-0.6	-0.640		-0.603	
-0.8	-0.810		-0.801	
-1.0	-1.000		-1.000	

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