Fundamentals of Dynamics (13)

Department of Mechanical and Control Engineering

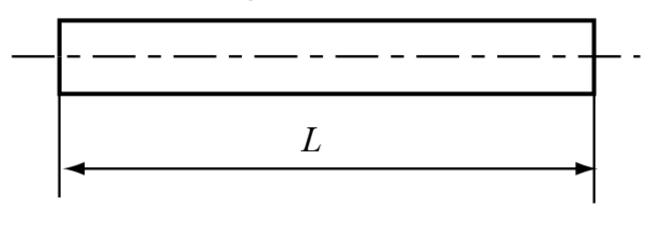
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Multi-degree-of-freedom Vibration System (2)

Analytical Solution

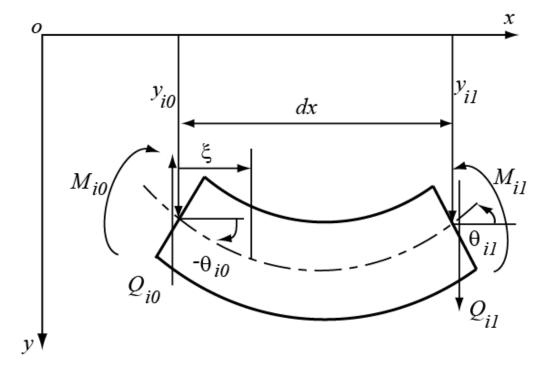
Bending Vibration of a Beam (3)

 ρ, A, E, I

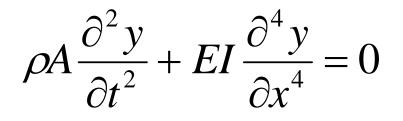


Uniform Beam

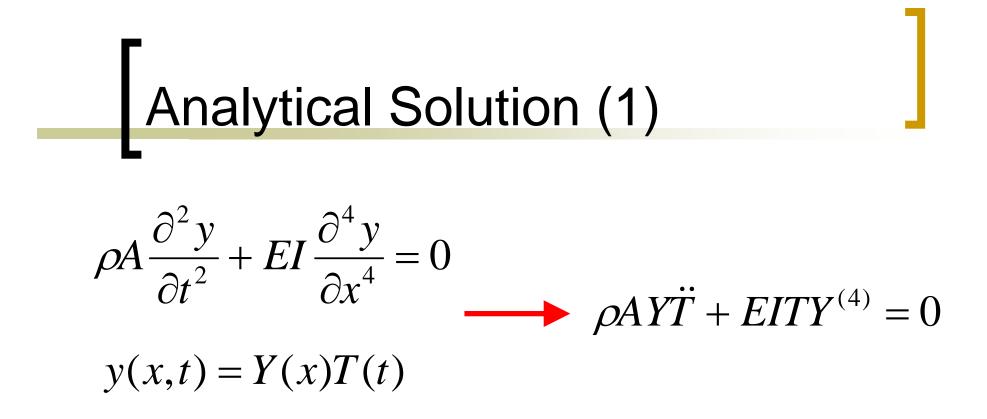
Bending Vibration of a Beam (4)



Equation of Motion



Coordinates and acting forces on an infinitesimal element



$$\frac{\ddot{T}}{T} = \frac{-EIY^{(4)}}{\rho AY} \equiv -\omega^2$$

a)
$$Y^{(4)} - a^4 Y = 0$$
 where $a^4 = \omega^2 \frac{\rho A}{EI}$
 $Y(x) = Y_1 e^{ax} + Y_2 e^{-ax} + Y_3 e^{iax} + Y_4 e^{-iax}$
 $Y(x)$: Eigen function

a is determined from boundary conditions of Y(x)

 a_i : i-th solution of the characteristic equation

Analytical Solution (3)

$$Y_{i}(x) = Y_{i1}e^{a_{i}x} + Y_{i2}e^{-a_{i}x} + Y_{i3}e^{ia_{i}x} + Y_{i4}e^{-ia_{i}x}$$
$$\{Y_{i1} Y_{i2} Y_{i3} Y_{i4}\} = \{\alpha_{i} \alpha_{i}y_{2i} \alpha_{i}y_{3i} \alpha_{i}y_{4i}\}$$

Orthogonality of eigen functions

$$\int_0^L Y_i(x)Y_j(x)dx = \begin{cases} 0 & i \neq j \\ \beta & i = j \end{cases}$$

Either α_i or β can be given arbitrarily.

(b)
$$\ddot{T}_i + \omega_i^2 T_i = 0$$

 $T_i(t) = T_i(0)\cos(\omega_i t) + \frac{\dot{T}_i(0)}{\omega_i}\sin(\omega_i t)$
where $\omega_i^2 = \frac{EI}{\rho A}a_i^4$

(c) General Solution

$$y(x,t) = \sum_{n=1}^{\infty} Y_i(x)T_i(t)$$

Analytical Solution (6)

(d) Initial Value Response y(x,0) and $\dot{y}(x,0)$ are given.

The value of β is set as 1.

$$T_{i}(0) = \int_{0}^{L} y(x,0)Y_{i}(x) dx$$
$$\dot{T}_{i}(0) = \int_{0}^{L} \dot{y}(x,0)Y_{i}(x) dx$$