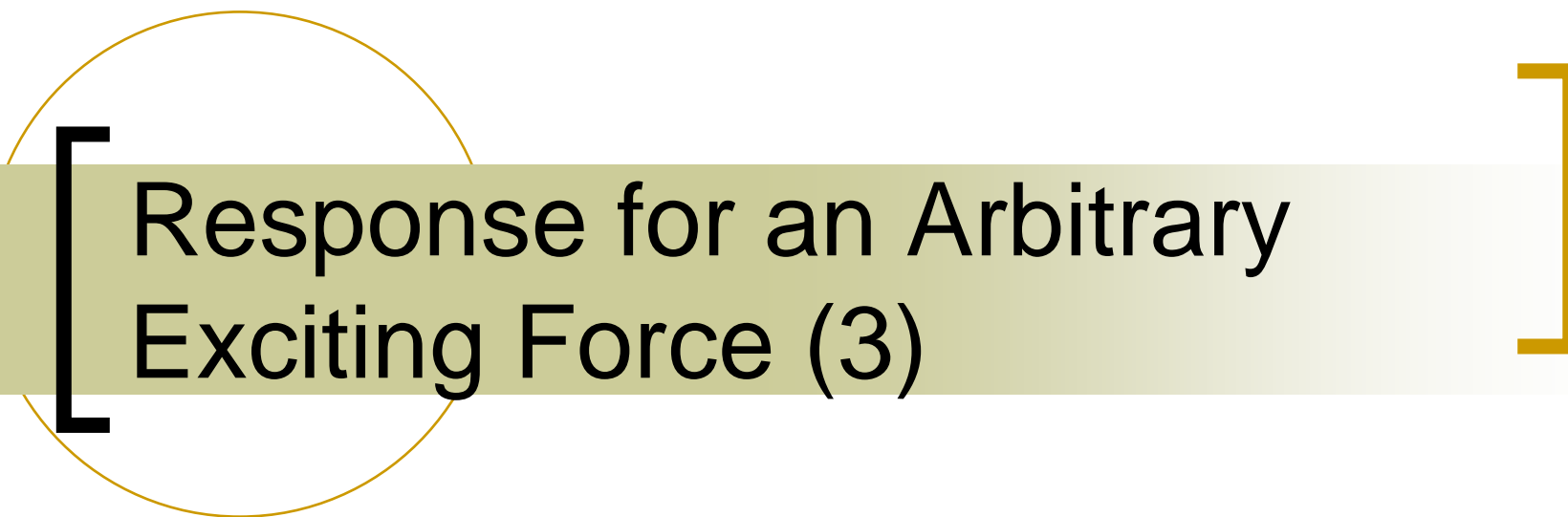




Mechanical Vibration I (14)

Department of Mechanical and
Control Engineering

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Response for an Arbitrary Exciting Force (3)

[Impulse response function (1)]

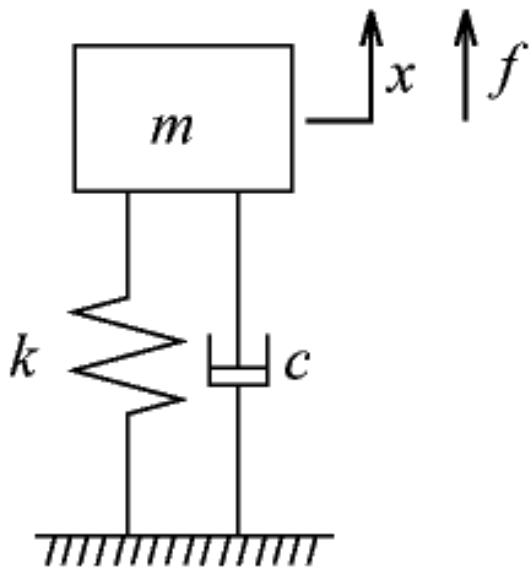


Fig.1 Damped one degree-of-freedom vibrationsystem with force excitation

Equation of motion

$$m\ddot{x} + c\dot{x} + kx = f$$

Unit impulse exciting force

$$f(t) = \delta(t)$$

Dirac's delta function

[Impulse response function (2)]

Table 1 Change of the states of the system with the unit impulse

Time	Momentum	Velocity	Displacement
$t = 0$	$m\dot{x} = 0$	$\dot{x} = 0$	$x = 0$
$t = \epsilon$	$m\dot{x} = 1$	$\dot{x} = 1/m$	$x = 0$

$$x(t) = e^{-\zeta\omega t} \frac{1}{m\omega_d} \sin \omega_d t \equiv h(t)$$

$h(t)$: Impulse response function

[Transient response (2-1)]

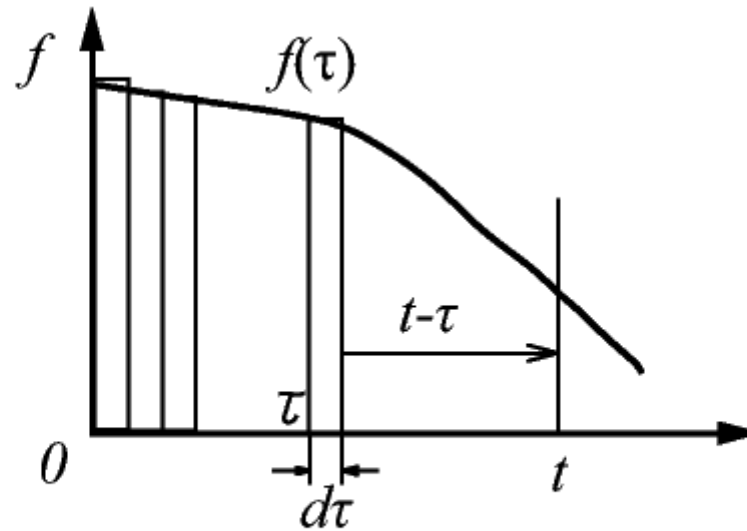


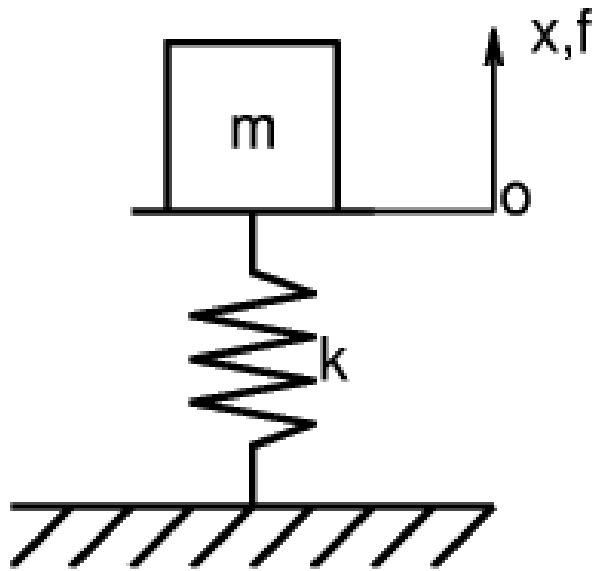
Fig.3 Decomposition of the exciting force into impulses

Transient response

$$x(t) = \int_0^t h(t - \tau) f(\tau) d\tau$$

[Transient response (2-2)]

Example



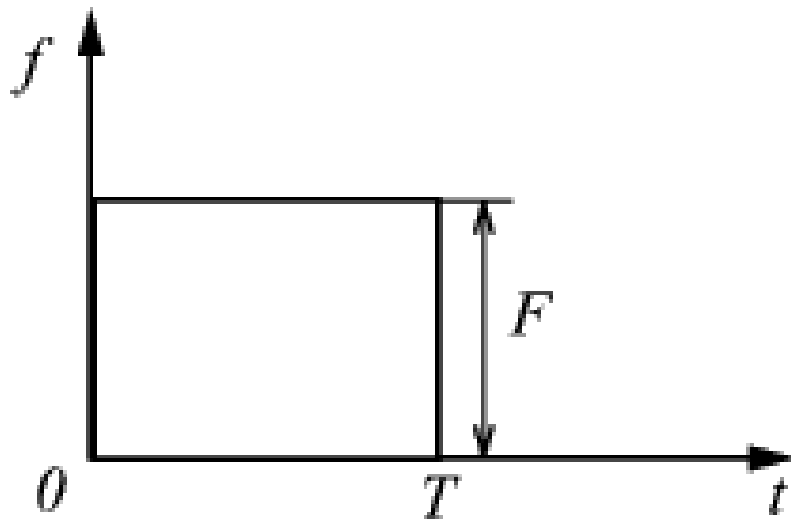
Impulse response function

$$h(t) = \frac{1}{m\omega_n} \sin \omega_n t$$

Fig.4 Undamped one degree-of-freedom vibration system

[Transient response (2-3)]

Example



Exciting force

$$f(t) = \begin{cases} 0 & (t < 0) \\ F & (0 \leq t \leq T) \\ 0 & (t > T) \end{cases}$$

Fig.5 Exciting force

[Transient response (2-4)]

Example

$$x(t) = \begin{cases} 0 & (t < 0) \\ \int_0^t Fh(t - \tau) d\tau & (0 \leq t \leq T) \\ \int_0^T Fh(t - \tau) d\tau & (t > T) \end{cases}$$