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One-degree-of-freedom Vibration System

### Actual vibration system (1)



(a) Tall Building



(b) Tire and suspension



(c) Simple pendulum

Fig.1 Actual vibration systems

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### Actual vibration system (2)



In many cases, the first mode is dominant in the free vibration.

Most vibration systems can be modeled as one degree-offreedom vibration system.

Multi-degree-of-freedom system can be represented as a superposition of one-degree-of-freedom vibration systems.

Fig.2 Example of free vibration

### Analytical model



Fig.3 Analytical model of one-degree-of-freedom vibration system

## Deriving the equation of motion (1)

#### Coordinate system

- The origin of the displacement should be placed on <u>the</u> <u>equilibrium point</u>.
- The direction of the force should agree with that of the displacement. Both of the Case 1 and 2 in Fig.4 are acceptable.



Fig.4 Coordinate system

# Deriving the equation of motion (2)

#### Reaction Force

- Reaction force of the damper is proportional to the velocity of the mass and its sign is minus.
- Reaction force of the spring is proportional to the displacement of the mass and its sign is minus.



Fig.5 Free body and acting force

Deriving the equation of motion (3)

#### d'Alembert's principle (1)

The sum of the differences between the generalized forces acting on a system and the time derivative of the generalized momentum of the system itself along an infinitesimal displacement compatible with the constraints of the system (a virtual displacement), is zero.

## Deriving the equation of motion (4)

#### d'Alembert's principle (2)

$$\left\{F - \frac{d(mv)}{dt}\right\}\delta x = 0$$

where 
$$F = -c\dot{x} - kx + f$$

Thus, the equation of motion is represented as the following.

$$m\ddot{x} + c\dot{x} + kx = f$$